Temperature-amplitude spectrum for early full-field vibration-fatigue-crack identification

Martin Česnik^a, Janko Slavič^{a,*}

^a University of Ljubljana, Faculty of Mechanical Engineering, Aškerčeva 6, 1000 Ljubljana, Slovenia

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Abstract

A dynamic structure under vibration loading within its natural frequency range can experience failure due to vibration fatigue. Understanding the causes of such failure requires pinpointing the initiation time and location of fatigue cracks, tracking their propagation, and identifying the frequency range of critical stress responses. This research introduces a novel, thermoelasticity-based method – the Temperature-Amplitude Spectrum (TAS) method – for earlystage, full-field, and non-contact crack detection that operates during uninterrupted vibration testing. This method leverages high-speed infrared imaging to analyze the specimen's temperature-amplitude spectrum, capturing comprehensive crack-related information, including initiation and propagation, in real time. Experimentally validated on both 3D-printed polymer and aluminum specimens, the TAS method accurately identified crack locations and paths without complex adjustments to the experimental setup or data processing. This new approach advances vibration-fatigue testing by enabling reliable, high-resolution crack detection and analysis while remaining computationally efficient.

^{*}Corresponding author. Tel.: +386 14771 226. Email address: janko.slavic@fs.uni-lj.si

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- thermoelasticity theory
- high-speed IR imaging
- temperature-amplitude spectrum
- early full-field crack identification

1 1. Introduction

Vibration fatigue is recognized as one of the most common failure mecha-2 nisms encountered during environmental vibration testing [1-3]. Over the past 3 decade, this field has garnered significant attention within the scientific community. As a result, vibration fatigue analysis has evolved beyond stationary Gaussian excitation and uniaxial stress conditions [4]. Current approaches now incorporate non-Gaussian [5-10], non-stationary [11-13], and multi-axis excita-7 tions [14, 15], along with their respective stress responses [16–19]. Additionally, 8 these studies now consider nonlinear dynamic systems [20–22]. Beyond evaluation and control across a broader range of signal types [23–25], recent research 10 has explored the influence of material properties on vibration fatigue. This in-11 cludes investigations into single-crystal superalloys [26] and various 3D-printed 12 metals [27] and polymers [28, 29], which can also exhibit frequency-dependent 13 fatigue parameters [30]. 14

Vibration fatigue is commonly estimated using spectral methods [31]. However, despite the accuracy of these methods, experimental vibration tests on actual structures remain necessary, as certain vibration loads can still lead to structural vibration-fatigue failure [32]. In such instances, key information for

enhancing a structure's vibration resilience includes the crack initiation time, lo-19 cation, severity, and the critical excitation/response frequency. Since vibration 20 testing applies a high load rate to the structure, any interruptions for dam-21 age inspection can significantly extend the testing duration [33]. Additionally, 22 inspecting for potential cracks typically requires dismounting, inspecting, and 23 remounting the structure, potentially altering its boundary conditions. There-24 fore, to maximize insights gained from vibration tests that result in fatigue 25 failure, it is essential to employ a real-time, non-intrusive, non-contact, and 26 full-field method for detecting damage. 27

Monitoring vibration fatigue can be accomplished using methods such as 28 vibration-based approaches [34, 35], digital image correlation (DIC) [36], and 29 thermography [37]. Starting with vibration-based methods: crack detection 30 typically involves observing changes in modal parameters [38] by evaluating 31 frequency-response functions. For example, Janeliukstis et al. [39] demonstrated 32 the effectiveness of the modeshape-curvature square method for crack localiza-33 tion. Gupta and Das [40] further enhanced this approach's accuracy by applying 34 a neural network trained with numerical models to extract error-free frequency-35 response data. For complex structures, the modal-strain-energy-index method 36 has shown high localization accuracy, as numerically shown by Zhang et al. [41]. 37 Bao et al. [42] also demonstrated precise localization using a multiple signal clas-38 sification (MUSIC) algorithm with the guided-wave method, highlighting early 30 crack detection and full-field observations. Although vibration-based methods 40 offer reliable crack detection, localization, and near-real-time monitoring, they 41 are somewhat limited in versatility, constraining their applicability across vari-42 ous products. 43

Conversely, the DIC method provides more flexibility and true full-field observation. Risbet *et al.* [43] showed that DIC could detect small strains under cyclic fatigue loading as early as 2010. Later, Kovarik [44] demonstrated DIC's capability in damage detection by monitoring strain fields during vibration tests

using low-speed cameras with a lock-in approach. Recently, Sun et al. [45] em-48 ployed a high-speed camera to detect damage at high load-cycle rates, as seen in 49 vibration fatigue. Zanarini [46] used DIC to obtain full-field frequency-response 50 functions to establish defect acceptance criteria. However, DIC has two main 51 drawbacks. First, obtaining strain requires double spatial differentiation, which 52 increases noise levels [47]. Second, the high computational demand for DIC, 53 particularly at high spatial resolutions, can be a limitation. These drawbacks 54 are typically not present in thermographic approaches. 55

Thermographic damage assessment using infrared (IR) imaging is a well-56 established [48], non-contact technique with extensive applications, often ap-57 plied in combination with thermoelastic stress analysis (TSA). D'Accardi et al. 58 [49] presented conductive-thermography technique for non-destructive crack de-59 tection. Additionally, Zhu et al. [50] and Bercelli et al. [51, 52] applied infrared 60 thermography to assess fatigue crack growth in metals, exploring the effects 61 of heat treatments and stress ratios on crack propagation and closure. Mean-62 while, Ricotta and Meneghetti [53], Amjad et al. [54], and Middleton et al. [55] 63 explored real-time, energy-based, and cost-effective thermographic monitoring 64 systems, emphasizing practical applications for detecting fatigue in materials 65 and large-scale structures. Thermoelastic approach is widely used for assess-66 ing the Paris' law [56], detecting damage in composite materials [57-59] and 67 for analyzing structural damping [60-62], though it has limitations when large 68 displacements are involved [63]. 69

Regardless, thermographic damage assessment continues to gain research interest, especially for high-rate loading conditions. Wei *et al.* [64] introduced a vibro-thermography method that uses an IR camera to identify fatigue cracks in specimens excited by a piezoelectric transducer in the ultrasonic range. Recently, Cai *et al.* [65] developed a method for monitoring fatigue damage in steel specimens during vibration testing, using a low-speed IR camera to observe heat generated by the specimen's resonant response at approximately 200

Hz. When subjected to vibration loads, temperature changes occur at faster 77 rates, requiring high-speed IR imaging for accurate monitoring. For instance, 78 Capponi et al. [66] estimated fatigue damage in a Y-shaped specimen under 79 random-signal multi-axial excitation, while Zaletelj et al. [47] applied the ther-80 moelastic principle to identify strain modeshapes in metal beams, capturing IR 81 data at 5000 frames per second. Recently, Sonc et al. [67] showed that ther-82 moelasicity principle can also be used as a criterion for vibration fatigue under 83 multiaxial loading. 84

Building on these advancements, the thermoelasticity approach presents a promising non-contact and non-invasive method for crack detection during vibration fatigue. This study addresses this potential by introducing a temperatureamplitude-spectrum (TAS) method for early crack detection using high-speed IR imaging. The TAS method is non-contact, computationally efficient, and non-intrusive, facilitating continuous, accurate, and early crack detection during vibration fatigue testing without interrupting the process.

This manuscript is structured as follows. Section 2 provides an overview of the physical principles underlying structural dynamics, vibration fatigue, and thermoelasticity. The novel TAS method for identifying vibration-fatigue cracks is introduced in Section 3. Section 4 details the experimental setup, testing procedure, and the range of specimens used. In Section 5, the results of crack identification using the TAS method are presented for aluminum and 3D-printed polymer specimens. Finally, conclusions are discussed in Section 6.

⁹⁹ 2. Theoretical background

This section initially presents the fundamental principles governing vibration fatigue in the context of kinematic (base) excitation [68]. Following this, a correlation is established between the structure's stress response and its temperature field, offering real-time insights into the structural condition as observed through infrared (IR) imaging.

¹⁰⁵ 2.1. Vibration fatigue of base-excited structures

Vibration fatigue arises when the frequency spectrum of the excitation aligns with the natural frequencies of the structure. If the real dynamic structure can be assumed to be linear and can be discretized to an *N*-degrees-of-freedom system, its governing equations of motion in the case of kinematic (base) excitation are formulated as [68]:

$$\mathbf{M}\ddot{\mathbf{z}} + \mathrm{i}\,\mathbf{D}\,\mathbf{z} + \mathbf{K}\,\mathbf{z} = -\mathbf{M}\,\mathbf{b}\,\ddot{y},\tag{1}$$

where \ddot{y} represents the acceleration of the base, z denotes the vector of relative displacements between the structure and the base, and b indicates the directional vector linking the structure's generalized coordinates with the direction of the base movement. **M**, **D** and **K** are the mass, damping and stiffness matrices, respectively. In Eq. (1), hysteretic damping is assumed.

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Solving the eigenvalue problem on the left-hand side of Eq. (1) yields the structure's natural frequencies, ω_r , damping ratios, η_r , and a modal matrix, Φ [69]. The matrix Φ consists of N mass-normalized modeshapes, $\phi_r =$ $[\phi_{r,1} \ \phi_{r,2} \ \cdots \ \phi_{r,N}]$. Next, a vector of mode-participation factors $\Gamma = \Phi^{T} \mathbf{M} \mathbf{b}$ [70] is introduced. Based on the linear relationship between the displacement modeshapes ϕ_r , strain modeshapes $\varepsilon \phi_r$ and stress modeshapes $\sigma \phi_r$ [71, 72], the stress response at the k-th stress degree-of-freedom can be expressed as [28]:

$$\sigma_k(\omega) = \sum_{r=1}^N \frac{\Gamma_{r\sigma}\phi_{r,k}}{\omega_r^2 - \omega^2 + i\eta_r\omega_r^2} \, \ddot{y}(\omega) = H_{\sigma\ddot{y},k}(\omega) \, \ddot{y}(\omega). \tag{2}$$

In Eq. (2), $H_{\sigma \ddot{y},k}(\omega)$ represents the structure's transmissibility, describing the influence of the base kinematics $\ddot{y}(\omega)$ on the stress response $\sigma_k(\omega)$ within the structure. Given the known power spectral density (PSD) [73] of the excitation, $G_{\ddot{y}\ddot{y}}(\omega)$, the stress response at the k-th stress degree-of-freedom can be written as:

$$G_{\sigma\sigma,k}(\omega) = |H_{\sigma\ddot{y},k}(\omega)|^2 G_{\ddot{y}\ddot{y}}(\omega).$$
(3)

From the perspective of vibration fatigue, dynamic structures accumulate 129 damage due to their dynamic stress responses. The impact of a material's fatigue 130 characteristics is quantified using Basquin's equation, $\sigma = S_f N^b$ [74], where S_f 13 denotes fatigue strength and b represents the fatigue exponent. Typically, the 132 linear damage accumulation rule is applied, where the damage D of a single 133 stress-load cycle is defined as $D = 1/N(\sigma)$. On a macro scale, structural failure 134 is recognized when D = 1. To estimate the damage accumulation rate d in 135 the fatigue zone of a dynamically excited structure, various frequency counting 136 methods can be applied [31]. When a single, distinct modeshape is excited, the 137 narrowband method provides reliable results, even with its simplicity [73]: 138

$$d^{\rm NB} = \frac{\left(\sqrt{2\,m_0}\right)^k}{2\,\pi\,C} \sqrt{\frac{m_2}{m_0}} \,\Gamma\left(1 + \frac{k}{2}\right),\tag{4}$$

where k = -1/b, $C = S_f^{-1/b}$, Γ denotes the Gamma function and m_0 , m_2 are the moments of the one-sided stress-response PSD $G_{\sigma\sigma,k}(\omega)$ (Eq. (3)).

¹⁴¹ 2.2. Thermoelasticity principle

The principle of thermoelasticity is founded on the interaction between a solid structure's mechanical and thermodynamic responses. This principle applies the fundamental laws of continuum mechanics alongside the first and second laws of thermodynamics. Assuming a fully reversible adiabatic process, the governing equation of thermoelasticity, which relates the stress field to the temperature field, is expressed as follows [75, 76]:

$$\rho C_{\sigma} \frac{\dot{T}}{T} = -\left[\alpha + \left(\frac{\nu}{E^2} \frac{\partial E}{\partial T} - \frac{1}{E} \frac{\partial \nu}{\partial T}\right) s\right] \dot{s} + \left(\frac{(1+\nu)}{E^2} \frac{\partial E}{\partial T} - \frac{1}{E} \frac{\partial \nu}{\partial T}\right) \boldsymbol{\sigma}_p \, \dot{\boldsymbol{\sigma}}_p,$$
(5)

where ρ denotes the density, C_{σ} is the specific heat at constant stress, and T is the absolute temperature. Additionally, α denotes the coefficient of linear expansion, E is Young's modulus, ν is Poisson's ratio, σ_p is the principal stress tensor, and s is the first stress invariant. In a uniaxial stress field, where $s = \sigma_1$ and $\sigma_2 = \sigma_3 = 0$, Eq. (5) simplifies to:

$$\rho C_{\sigma} \frac{\dot{T}}{T} = -\left(\alpha - \frac{1}{E^2} \frac{\partial E}{\partial T} \sigma_1\right) \dot{\sigma}_1.$$
(6)

If the principal stress load is sinusoidal, expressed as $\sigma_1(t) = \sigma_{1,m} + \sigma_{1,a} \sin(\omega t)$, the linearized solution of Eq. (6) about the reference temperature T_0 becomes [77]:

$$\frac{\rho C_{\sigma}}{T_0} \Delta T(t) = -\left(\alpha + \frac{1}{E^2} \frac{\partial E}{\partial T} \sigma_{1,\mathrm{m}}\right) \sigma_{1,\mathrm{a}} \sin(\omega t) + \frac{1}{4E^2} \frac{\partial E}{\partial T} \sigma_{1,\mathrm{a}}^2 \cos(2\omega t).$$
(7)

¹⁵⁷ Under the assumption of an adiabatic process, the temperature within the ¹⁵⁸ observed control volume – subjected to harmonic stress excitation – oscillates ¹⁵⁹ with both the fundamental excitation frequency ω and its second harmonic 2ω . ¹⁶⁰ For high-cycle fatigue conditions, resulting from the structure's dynamic re-¹⁶¹ sponse, the observed temperature range T remains narrow. In this scenario, ¹⁶² variations in Young's modulus with respect to temperature $\partial E/\partial T$ are negligi-¹⁶³ ble. Consequently, Eq. (7) simplifies to:

$$\Delta T(t) = -\frac{\alpha T_0}{\rho C_\sigma} \sigma_{1,a} \sin(\omega t) = K_m \sigma_{1,a} \sin(\omega t), \qquad (8)$$

where $K_m = -(\alpha T_0)/(\rho C_{\sigma})$ is the thermoelastic coefficient [75]. The relationship between stress response and temperature at the observed location can be generalized as $\Delta T(t) = K_m \sigma(t)$ when the dynamic structure is excited in a single, well-separated mode shape, due to the narrow-band nature of the stress response (Eq. (2)).

In applying the thermoelasticity principle to the identification and quantification of fatigue cracks, the adiabatic condition can be assumed to hold across the
entire observed (visible) surface, provided the observation times are sufficiently
short.

As a fatigue crack develops and penetrates the material, the stresses on the observed surface near the crack are reduced, leading to an absence of temperature variations due to the thermoelastic effect, see Fig. 1. However, temperature fluctuations still occur at the crack tip. At this stage, the temperature distribution near the crack is governed by the heat-diffusion equation:

$$\frac{\partial T}{\partial t} = \frac{k}{\rho C_{\sigma}} \left(\frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right),\tag{9}$$

where k is the thermal conductivity.

¹⁷⁹ 3. Temperature-amplitude-spectrum (TAS) method

This section introduces the TAS method for early, full-field crack identification in structures experiencing vibration fatigue. This approach relies on capturing the complete temperature field across the surface within the fatigue zone, using a sampling frequency that significantly exceeds the structure's natural frequency.

The main concept of this methodology is illustrated in Fig. 1 and described as follows. According to Eq. (3), when a dynamic structure is excited by a random wide-band signal, it exhibits amplified stress-response amplitudes at its natural frequencies (see Fig. 1, left). Based on thermoelasticity theory (Eq. (8)), the temperature response on the structure's surface mirrors the stress response, scaled by a thermoelastic constant. Consequently, a specific natural frequency and its associated stress modeshape can be isolated within the fatigue zone.

During broad-band excitation, the stress response near a natural frequency appears as a narrowband signal [73]. The TAS method, which employs an infrared (IR) camera, provides a full-field approach that captures spatial information across the surface. As shown in Fig. 1, the stress near the crack diminishes, as maximum stress propagates with the crack tip into the structure. This reduction is also evident in the lowered maximum value of the temperature-amplitude ¹⁹⁸ spectrum, obtained through a fast Fourier transform [78]. The concept is fur-¹⁹⁹ ther illustrated in Fig. 1.



Figure 1: Crack detection concept: the temperature signal during the vibration test due to the thermoelastic effect and its changes in the presence of a crack.

In the following, the concept described above is generalized. Consider a 201 measured temperature time series for the *i*-th pixel, represented as $T_i(t_n) =$ 202 $T_i(n \cdot \Delta t)$, where $n = 0, \ldots, N - 1$. Here, Δt denotes the time step between 203 sequential frames, and N is the total length of the time series T_i , *i. e.* the number 204 of frames in the observed recording. Given the temperature time series $T_i(t_n)$ 205 for the *i*-th pixel, the temperature-amplitude spectrum $\hat{T}_i(\omega)$ can be obtained 206 using a discrete Fourier transform [73], expressed as $\hat{T}_i(\omega_k) = |DFT(T_i(t_n))|$, 207 where $\omega_k = k \cdot \Delta \omega$, $k = 0, \dots, N - 1$, and $\Delta \omega = 2\pi/(N\Delta t)$. The temperature-208 amplitude spectrum (TAS) for the i-th pixel is defined as: 209

$$\hat{T}_{i,AS} = \max(\hat{T}_i(\omega_k)), \text{ where } \omega_k \in (\omega_{\min}, \omega_{\max}),$$
 (10)

where ω_{\min} and ω_{\max} denote the frequency range of excitation PSD $G_{ijij}(\omega)$. Two conditions must be satisfied for accurate analysis. First, the structure's critical natural frequency ω_r should fall within the excitation frequency range $(\omega_{\min}, \omega_{\max})$. Second, the sampling frequency $f_s = 1/\Delta t$ must be high enough to ensure the Nyquist frequency $f_s/2$ exceeds $\omega_{\text{max}}/2\pi$. It is important to note that the frequency of maximum magnitude may vary across different pixels; however, these peak frequencies are generally expected to cluster around the structure's critical natural frequency. The definition of the estimator $\hat{T}_{i,\text{AS}}$ in Eq. (10) highlights the computational efficiency of the TAS method. Despite its simplicity, this method provides robust crack detection capabilities, as demonstrated in the experimental results in Sec. 5).

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To compare the TAS method with the conventional thermography approach, which employs low-frame-rate infrared (IR) imaging, this study introduces an alternative image-processing technique. Due to the long exposure times associated with low-frame-rate imaging [65, 66], the temperature time series of the *i*-th pixel is compressed into an average temperature estimator:

$$T_{i,\text{TG}} = \frac{1}{N} \sum_{n=1}^{N} T_i(t_n).$$
 (11)

²²⁷ In this study, this approach is referred to as the "Thermography method".

228 4. Experimental research

The experimental framework developed to demonstrate the feasibility of the TAS method is outlined as follows. First, the design of the test specimens and the configuration of the experimental setup are described in detail. This is followed by a discussion of the variations among the tested specimens. Finally, preliminary experimental results are presented to provide an initial assessment of the effectiveness of the TAS method.

235 4.1. Experimental setup

The experimental setup follows a standard approach for vibration testing using an electrodynamic shaker, with the addition of a high-speed IR camera for enhanced measurement capabilities, as illustrated in Fig. 3a). The design of

the specimens, adapted from [28] and detailed in Fig. 2(a) consists of a fixation 239 areas, a notch area and an inertial mass. With a proposed simple design it is 240 possible to achieve well-separated mode shape, a tuneable natural frequency, 241 and an accessible fatigue zone (Fig. 2(b)) with a near-uniaxial stress field. To 242 confirm the applicability of TAS methodology, the specimens were produced 243 from PLA by 3D printing and from aluminum by conventional machining. The 244 specimens were mounted to the shaker's armature with an M6 bolt and a 3-mm-245 thick aluminum plate. Additionally, to allow free movement of the specimen, the 246 lower fixation surface was shimmed with a 3-mm aluminum plate, as depicted 247 in Fig. 3b). For conducting vibration-fatigue tests the base excitation is defined 248 by an acceleration random-signal PSD profile $G_{ijij(\omega)}$, Eq. (3). An LDS V555 249 electrodynamic shaker was used, and the PSD acceleration profiles were flat-250 shaped (Fig. 1) with adjustable amplitude and frequency ranges. To monitor 251 the specimen's response, a response accelerometer was employed, as shown in 252 Fig. 3b). 253

For high-speed infrared (IR) imaging, a Telops FAST m3K camera was used. The camera has a specified Noise Equivalent Temperature Difference (NETD) of 32 mK; however, as demonstrated by Zaletelj *et. al* [47], significantly lower noise levels can be achieved through optimized signal processing in the frequency domain. The IR camera was fitted with a Telops 1X microscopic lens with a fixed focal distance of 26 cm. The experimental setup and its implementation are shown in Fig. 3.

²⁶¹ 4.2. Specimen overview and testing conditions

Two types of specimens were evaluated: the 3D-printed polylactic-acid (PLA) specimens and aluminum specimens. The 3D-printed PLA specimens were produced in two orientations (x and y, see Fig. 2) with an inertial mass length of L = 24 mm. These specimens were fabricated using a Prusa i3 MK3S+ 3D printer with a 0.4-mm nozzle diameter, a nozzle temperature of 215°C, and



Figure 2: Specimen design; (a) specimen geometry with adopted coordinate system, (b) manufacutured specimen with denoted observed fatigue zone, (c) zoomed-in fatigue zone for specimen 3D-printed in y direction and (d) in x direction.



Figure 3: Experimental setup; (a) a schematic representation with an electrodynamic shaker, mounted specimen and a high-speed IR camera, (b) an actual setup.

No.	Material	$L \; [\rm{mm}]$	Print direction	Surface preparation
1	PLA	24	y	/
2	PLA	24	x	/
9	Aluminum	40	/	Fine ground,
ა	AIUIIIIIIIIIIIII	40	/	black color spray coating
4	Aluminum	40	/	/

Table 1: Overview of the tested specimens.

varying printing speeds: 25 mm/s for the external perimeter, 45 mm/s for the 267 internal perimeter, and 80 mm/s for infill. No additional surface preparation 268 was applied to the 3D-printed specimens. Fig. 2(b) a 3D-printed specimen with 269 a denoted area of observation during vibration testing; an enlarged view of the 270 observed area is given in Figs. 2(c) and 2(d) for 3D printing in y and in x di-271 rection, respectively. The aluminum specimens were manufactured from 6026 272 aluminum alloy using water-jet cutting, with an inertial mass length of L = 40273 mm. These specimens underwent additional surface post-processing, including 274 fine grinding and painting with black spray to enhance IR imaging contrast. A 275 summary of the specimen types and specifications is provided in Tab. 1. 276

To determine the PSD acceleration profile parameters for each test specimen, two main guidelines were followed: first, the natural frequency of the specimen needed to be continuously excited throughout the test; second, complete fatigue failure was expected to occur within 10 to 30 minutes from the start of the vibration test. The first natural frequencies of the undamaged specimens, along with the specific testing conditions for each, are provided in Tab. 2.

The high-speed camera captured IR images at a rate of 2400 frames per second, with a spatial resolution of 320×265 pixels, covering a surface area of 10.0×8.3 mm². IR imaging was automatically triggered at constant time intervals, capturing 1200 frames per trigger, resulting in a recorded duration of

No.	$f_1 [\mathrm{Hz}]$	Excited freq. range [Hz]	PSD level $[(m/s^2)^2/Hz]$
1	226	[150, 400]	0.4
2	248	[150, 350]	2.0
3	505	[450, 550]	5.4
4	530	[450, 550]	4.0

Table 2: Specimens' natural frequencies and testing conditions.

0.5 seconds per interval. The control and response accelerometers sampled data
at a frequency of 25.6 kHz, with a 10-second averaging period applied to obtain
the specimen's frequency-response function.

²⁹¹ 4.3. Experimental results and IR image processing

According to thermoelasticity theory (Sec. 2), the temperature response of 292 the specimen in the fatigue zone is primarily governed by the stress-response 293 frequency. For the tested specimens, and as indicated in Eqs. (2) and (3), this 294 frequency corresponds to the specimen's first natural frequency. The temper-295 ature response of specimen no. 1, both in the time and frequency domains, is 296 shown in Figs. 4(a) and 4(b) after 100 seconds of vibration testing. The mea-297 sured temperature data refer to pixel no. 45267, marked in Fig. 4(c) as the pixel 298 with the highest $T_{i,TG}$ value within the 0.5-second recording interval. The pre-299 liminary measurement of specimen no. 1's natural frequency (Tab. 1) demon-300 strates that the observed temperature response, specifically the temperature-301 amplitude spectrum displayed in Fig. 4(b), aligns with the analytically predicted 302 response outlined in Eq. (8). Furthermore, the temperature-amplitude spectrum 303 in Fig. 4(b) shows no peak at the second harmonic of the specimen's natural 304 frequency, $2 \cdot f_1$. This observation supports the assumption of $\partial E / \partial T = 0$ in 305 Eq. (7). 306

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³⁰⁸ The temperature-amplitude spectrum in Fig. 4(b) shows noise within the

frequency range of the highest magnitudes. To address this, Welch's averaging was applied to the temperature signals, following the method outlined in [66], which enhances the signal-to-noise ratio in the frequency domain [79]. While this approach improved the noise profile, it did not significantly enhance the accuracy of crack identification. Therefore, to reduce the computational cost associated with the TAS method, a non-averaged amplitude spectrum approach was ultimately adopted.

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By calculating the TAS values as in Eq. (10) or 'Thermography' mean values 317 as in Eq. (11), a single representative value per pixel can be obtained for each 318 recording. These pixel values can then be visualized as a heat map, providing 319 a graphical representation of the entire recording. Figs. 4(c) and 4(d) show the 320 heatmaps of specimen no. 1 after 100 seconds of vibration testing, comparing the 321 TAS method with the Thermography method. The TAS method clearly offers 322 a more detailed visualization of the specimen's surface. In generating Fig. 4(d) 323 the amplitude spectra values were extracted around a frequency of 225 Hz. 324

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A response accelerometer was attached to the inertial mass of each specimen 326 to monitor its frequency-response function during vibration-fatigue testing. By 327 applying the least-squares complex-frequency-domain (LSCF) method [80] to 328 the measured frequency-response data, precise information on changes (i.e., de-329 creases) in the specimen's natural frequency and damping over the course of the 330 vibration test was obtained. Consequently, the TAS method should also capture 331 these shifts in the specimen's natural frequency. This prediction was validated 332 in the present study, as illustrated in Fig. 5, where the mean frequency of the 333 pixels' $\hat{T}_{i,AS}$ values is shown alongside the natural frequency values derived from 334 the response accelerometer. Fig. 5 further indicates that aluminum specimens 335 exhibit a lower scatter in the averaged frequency of maximal response compared 336 to polymer specimens. This reduced scatter may be partially due to the higher 337



Figure 4: Temperature response of the specimen no. 1; (a) time waveform for pixel no. 45267, (b) amplitude spectrum for pixel no. 45267, (c) heatmap of observed area obtained with Thermography method and (d) with TAS method.

natural frequency of the aluminum specimens, which leads to a greater number of load cycles within each 0.5-second observation interval, enhancing the
stationarity of the temperature signal. Conversely, the temperature signal of
the polymer specimens shows more non-stationarity within a single 0.5-second
recording, which can also be observed in the time-domain signal presented in
Fig. 4.

³⁴⁴ 5. Crack identification using the TAS method

The primary objective of the novel TAS method is to identify the spatial location of vibration-fatigue cracks at an early stage. Results from four tested



Figure 5: Comparison between (—) specimens' identified natural frequencies with a response accelerometer and (•) averaged frequency of $\hat{T}_{i,AS}$, graphs (a) - (d) refer to specimens no. 1 - no. 4, respectively.

specimens, listed in Tab. 1, are presented in Figs. 6 through 9. Each figure shows a decrease in the specimen's natural frequency alongside a photograph of the resulting cracks, with jet color-mapping applied in the crack area to enhance the crack's visibility. IR images processed with the TAS method are displayed above the frequency plots, while the Thermography method results are shown below. The testing times highlighted in Figures 6-9 were chosen based on crack changes detected by the TAS method.

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Examining the results for the 3D-printed specimens no. 1 and no. 2 (Figs. 6 355 and 7, respectively), it is clear that the TAS method offers significantly im-356 proved crack localization compared to the conventional Thermography method. 357 As expected (Sec. 3), cracks appear as areas of locally reduced temperature 358 in the TAS images, consistent with thermoelasticity theory (Eq. 7). As the 359 crack propagates, these zones of reduced thermoelasticity-induced temperature 360 expand accordingly. The final TAS image of specimen no. 1 (Fig. 6 at 1600 s), 361 obtained with the TAS method, aligns well with the actual final crack condition. 362

³⁶³ By contrast, the Thermography method provides less detailed results, primarily ³⁶⁴ identifying only the crack tips. This limitation in the Thermography method's ³⁶⁵ detection efficacy arises due to heat conduction from the internal stress con-³⁶⁶ centration at the crack tip to the specimen's surface, as governed by the heat ³⁶⁷ diffusion law (Eq. (9)). It should also be noted that a minor printing defect was ³⁶⁸ present at x = 3 mm and y = 3.5 mm in the observed area, which could have ³⁶⁹ falsely suggested an early crack at 60 seconds of testing.



Figure 6: Crack identification of specimen no. 1 (3D printed in y direction) during vibrationfatigue testing using the TAS method (upper section) and Thermography method (bottom section) in reference to the drop of the specimen's natural frequency and a final crack in the middle section.

The results for specimen no. 2 are presented in Fig. 7. The TAS method reliably detects the failure of two individual threads, each 0.2 mm wide, appearing after 945 seconds of vibration testing. This failure is minor enough that it does not result in any detectable change in the specimen's natural frequency. As the
vibration test continues, the TAS method clearly maps the propagation path of
all three initial cracks. In contrast, as with specimen no. 1, the Thermography
method provides significantly less detailed information on crack initiation and
propagation for specimen no. 2.





Figure 7: Crack identification of specimen no. 2 (3D printed in x direction) during vibrationfatigue testing using the TAS method (upper section) and Thermography method (bottom section) in reference to the drop of the specimen's natural frequency and a final crack in the middle section.

Additionally, the study included tests on 3D-printed specimens oriented in the z-direction (Fig. 2(a)), following the same experimental procedure. Crack detection using the TAS method on these specimens was less distinct than for those printed in the x- and y-directions, though it still yielded more accurate results than the Thermography method. The reduced effectiveness in identifying cracks is likely due to the obscured view of the crack initiation and propagation
path, which is concealed beneath subsequent filament layers (for further details,
see [28]).

388

The initial analysis of the aluminum specimens shown in Figs. 8 and 9 reveals 389 a similar decrease in natural frequency for both specimens. In general, the TAS 390 method reliably detects the vibration-induced fatigue crack in both cases. A 391 closer examination of the natural frequency drop indicates that crack initiation 392 is recognized earlier on specimen no. 3 (fine ground with black coating) than on 393 specimen no. 4 (water-jet cut with no coating). However, the spatial accuracy of 394 crack identification is higher for specimen no. 4 compared to no. 3. In addition 395 to specimens no. 3 and no.4, an additional type of aluminum specimen was 396 tested with a fine ground surface and no color coating. Crack detection on this 397 uncoated specimen was unsuccessful due to high surface reflectivity, which led to 398 incorrect measurements from the high-speed IR camera. Therefore, black color 399 spraying is recommended for crack detection on reflective surfaces, although 400 a slight increase in spatial uncertainty may be expected. It should also be 401 noted that uneven paint application can vary paint thickness, causing random 402 spots in the processed IR images (see Fig. 8). This issue can be effectively 403 mitigated by comparing image changes relative to the initial test image. In 404 contrast, the Thermography method failed to detect any cracks in the aluminum 405 specimens. This limitation is likely due to the low heat generation and high 406 thermal conductivity of aluminum. 407

Furthermore, the final crack condition in both specimens consists of multiple parallel cracks. While the TAS method reliably identifies the presence of these cracks, the accuracy in localizing the exact crack paths is reduced in such cases. This outcome aligns with expectations from thermoelastic theory, which primarily detects areas of reduced stress amplitude – areas that are often found between parallel cracks.



Figure 8: Crack identification of specimen no. 3 (aluminum with prepared surface) during vibration-fatigue testing using the TAS method (upper section) and Thermography method (bottom section) in reference to the drop of the specimen's natural frequency and a final crack in the middle section.

It is noteworthy that successful crack detection results were achieved with the novel TAS method on both polymer and aluminum specimens using the same image processing procedure, without any fine-tuning of the TAS method. Additionally, with a larger group of test specimens, the reliability of crack detection was consistently confirmed. In each case, a clear overview of the crack location was achieved, provided that the reflective effect was sufficiently minimized.

420 6. Conclusions

This study introduces a novel approach for identifying cracks due to vibration fatigue, utilizing the temperature-amplitude spectrum (TAS). The TAS method, based on thermoelastic theory and applied through high-speed IR imag-



Figure 9: Crack identification of specimen no. 4 (aluminum with surface from water-jet cutting) during vibration-fatigue testing using the TAS method (upper section) and Thermography method (bottom section) in reference to the drop of the specimen's natural frequency and a final crack in the middle section.

ing, enables the real-time detection of crack initiation and progression within
the specimen's fatigue zone without disrupting ongoing vibration tests. The
method's effectiveness was demonstrated across multiple specimens with natural frequencies around 210 Hz and 510 Hz, highlighting its versatility in varying
frequency conditions.

The established TAS method provides several distinct advantages. As a fullfield, non-contact approach, it achieves high spatial resolution (320x256 pixels) and can detect early-stage cracks as small as 0.2 mm, even before any shift in natural frequency occurs. This capability allows for precise crack localization and provides detailed insights into both crack propagation and the critical frequency ranges involved in excitation and response, which are essential for 435 assessing structural integrity under dynamic loading.

In terms of practical application, the TAS method proves as very adaptable. It performs reliably on both polymer and metal specimens without requiring adjustments of image processing procedures. The setup demands only a single measurement device, with minimal specimen surface preparation, making the method accessible for routine use. Additionally, its computational efficiency supports real-time monitoring during vibration-fatigue testing, which is crucial for early intervention and structural health monitoring.

Nevertheless, the TAS method has two primary limitations. First, reflective surfaces require a black coating to reduce interference in IR imaging. Second, the method's accuracy depends on maintaining an unobstructed view of the crack to capture clear temperature changes associated with crack initiation and propagation. Despite these limitations, the TAS method represents a significant advancement in non-destructive testing for vibration-fatigue assessment, combining practical ease with high sensitivity to early-stage damage.

450 Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could influence the work reported in this paper.

453 Data availability

⁴⁵⁴ Data will be made available on request.

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